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Modeling of Optimized Neuro-Fuzzy Logic Based Active Vibration Control Method for

Automotive Suspension

Mohammad Adom Safiullah

A Thesis Submitted to the Graduate Faculty of

GRAND VALLEY STATE UNIVERSITY

In

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Dedication

To my mother, Professor Showkat Ara Begum, for her support and presence in my life.

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I would like to thank my thesis supervisor Dr. Nicholas Baine first. He guided me when I felt lost and whenever I needed help he was there to support. I am especially grateful for his guidance on the literature writing, without his supervison this thesis would not be possible. I would also like to thank Dr. Shabbir Choudhuri for his help with this thesis. His expertise and suggestion was immensely helpful and made this thesis a better work. I also owe gratitude to Dr. Ryan Krauss for his encouragement during this work.

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Abstract

In this thesis, an active vibration control system was developed. The control system was developed and tested using a quarter car model of an adaptive suspension system. For active vibration control, an actuator was implemented in addition to the commonly used passive spring damper system. Due to nature of unpredictability of force required two different fuzzy inference system (FIS) were developed for the actuator. First a sequential fuzzy set was built, that resulted lower vertical displacement compared to basic damper spring model, but system had limited effect with disturbances of higher magnitude and continuous vibrations (rough road). To improve the performance of the sequential fuzzy set, the main fuzzy set was improved using an adaptive neuro fuzzy inference system (ANFIS). This model increased the performance substantially, especially for rough road and high magnitude disturbance scenarios. Finally, the suspension's spring constant and damping co-efficient was optimized using a genetic algorithm to further improve the vibration control properties to achieve a balance of both ride stability and comfort. The final result is improved performance of the suspension system.

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1. Introduction

Vibration isolation and control of a suspension is a popular topic of study in the automotive industry. In this research, a quarter-car model is studied along with active vibration control. An active element (actuator) is used to apply force on the car body to continuously control vibration, and a fuzzy logic controller is used to determine how much force is to be applied.

Quarter Car Model (QCM) is used to study the vibration on a car. QCM simplifies the study but provides a representative result for the vibration and corresponding impact on the car. For this general vibration study, the QCM consist of two springs and two dampeners, one for suspension and another representing the tire of the car [1]. For this active suspension model, an actuator was added to mitigate the impact of vibrations in the road disturbance on the displacement of the car between the mass of the car and the wheel. This QCM model is depicted in Figure 1 with the actuator.



Figure 1: Quarter Car Model

Here, m_1 is mass of quarter car, k_1 is spring constant of suspension, b_1 is dampening coefficient of suspension, x_1 is vertical displacement of car, m_2 is mass of wheel, k_2 is spring constant of tire, b_2 is dampening coefficient of tire, x_2 is vertical displacement of wheel, w is road disturbance and \underline{u} is the actuator force applied.

The force applied by the actuator is calculated using a Fuzzy Inference System (FIS). FIS is a human experience-based approach to make a computation that is not well defined mathematically [2]. However, the rule base for a traditional fuzzy logic controller is limited by the scope and capabilities of human experience and abilities. To work around this limitation, neural learning can be a useful method to improve upon the FIS model, with the system being trained to generate its own rule base based on a given set of training data. The combination of the two concepts is referred to as an Adaptive Neuro Fuzzy Inference System (ANFIS), which has a fuzzy set where the rule base is result of neural training based on data provided from human experience.

While neural learning is known to be inspired by the functioning of human brain, Genetic Algorithms (GA) are based on natural selection. The third method used to improve performance is GA optimization, which is based on survival of fittest principle. A genetic algorithm will be used to optimize the system being controlled. This works by creating multiple versions of the system and comparing their performance. The best designs survive and are mixed to create the next generation of designs. In each generation, GA optimization generates a new population keeping only the best designs to be used to create subsequent populations. Eventually, the algorithm will converge on an optimized solution with a better performance than any of the initial designs in the first population.

1.1. Objective

In this work, a second-order differential equation of forced damped vibration for a quarter car suspension model is used. The objectives of this research work are:

- Development of a state-space model for forced damped vibration
- Modelling of quarter car based on previous studies
- Developing a FIS model for active vibration control of a suspension
- Improving fuzzy rule base with ANFIS training
- Optimizing suspension system parameters using a GA to further improve performance

1.2. Scope

This work models an active suspension system using a quarter car model, where the active force calculation system was developed using FIS and ANFIS methodology and the performance was compared. For FIS, the Mamdani inference system was used. For ANFIS the Takagi-Sugeno method and hybrid method were used for training. Finally, a genetic algorithm was used for optimization of the actual system model design. The system was limited to two masses and parameters were constrained as there are realistic limitations.

2. Background and Literature Review

In various industries, noise and vibration control is an important concern. In the conventional spring damper system, springs store energy and dampeners dissipate the energy to reduce the effect of vibration. In a passive suspension system, there is no control of the spring and dampener; therefore, the vibration cannot be isolated and used to provide feedback. For the modern automobile vibration, researchers have improved upon the passive suspension through the development of active and semi-active vibration control studies. In semi-active suspension systems, vibration control is improved by changing the physical properties of suspension at certain stages. Alternatively, active suspension systems work by continuously monitoring and injecting the proper amount of force through an actuator attached with a conventional suspension.

The active control of vibration is a popular research topic, with much work being done using both theoretical and experimental studies [3]. The comfort and maneuverability of a vehicle depends heavily on the response of the suspension and its ability to reject disturbances (i.e. vibration control). Semi-active suspension performs adequately in most scenarios, but with the growing possibility of the driverless car in the near future and increasing demand of smoother vibration control, active vibration control is being explored as a better solution according to Pinhas Barak [4]. His work predicted that active vibration control will be able meet the demand of the future, as new developments make it more practical to implement. Dean Karnopp in his study used [5] the skyhook damper model for the active suspension. The basic Skyhook damper system only used damping to control the vibration, but it has its limitation in flexibility and operation in the robust system. Scott Ikenaga et al. developed an active suspension control for a full car model using a control system that combines filtered feedback and input decoupling transformation [6]. They used skyhook damping and mitigated the vibration by actively controlling the damping coefficient of a semi-active damper. For the nonlinear system, the skyhook damper has few limitations. For better flexibility and response Krtolica and Hrovat solved a fourth-order linear quadratic differential equation for a half-car model in order to find the optimal solution to vibration control [7].

For the nonlinear system, Ozgur Demir et al. used fuzzy logic combined with a PID controller for a suspension design of a half-car model [8]. Fuzzy logic improves the control of the nonlinear system, and in the robust nonlinear system, the fuzzy logic system can react fast and has great performance [9]. Qu Wenzhonga et al. have shown the advantage of fuzzy logic over filtered-X LMS algorithm [10]. They proved that use of fuzzy logic can be advantageous over an algorithm like filtered-X LMS, which is simple to use and requires low computational load but is more suitable for linear control problems. Shiuh-Jer et al. [11] have designed an adaptive fuzzy controller using the sliding mode controller, where a smaller rule base is required, but they implemented online learning to compensate the system's time-varying and nonlinear behavior. Jinpeng Li et al. [12] elaborated on the coupling of fuzzy logic and sliding mode controller where they designed an adaptive fuzzy system for a semi-active suspension system.

Implementation of online and different machine learning is also becoming common in vibration studies. M Soleymani et al. [13] used online learning to make the suspension system react with not only road conditions but also traffic conditions. Various machine learning and genetic algorithm are also being used to design robust fuzzy logic control systems. Wei-Yen Wang et al. [14] used neuro-fuzzy logic, and Tomonori Hashiyama et al. [15] used the genetic algorithm with fuzzy logic to control an active or semi-active suspension.

Neuro-fuzzy and genetic fuzzy algorithm-based suspension is more robust and efficient compared to the traditional fuzzy algorithms, but they are more computation heavy to train. In

either case, the fuzzy logic itself is computationally light and can be implemented on low-end suspension with little cost increase for application. In a complex non-linear system, the rule base often gets very convoluted. In this study, a sequential fuzzy set approach is explored instead of one monolithic fuzzy set; this provided scope to focus rules to specific conditions and allowed for an easy to comprehend and editable rule-base for the fuzzy set. However, after development, the sequential fuzzy set could be changed to a monolithic fuzzy set with no significant performance change.

2.1. Fuzzy Inference System (FIS)

In this work, a quarter car model was used as the plant to study fuzzy logic based active vibration control. Mamdani model for the fuzzy set was selected as it provides a simpler rule interface. Rules in a Mamdani set can be developed over human experience. This is in comparison to the mathematical rule base in Takagi-Sugeno among other fuzzy approaches. For membership functions, simplicity and effectiveness of isosceles triangles were used by Manu Sharma et al. [16]. They have studied the effect of the right-angled and isosceles triangle on Mamdani type Fuzzy control system. Grzegorz Filo [17] showed ways to use MATLAB/Simulink's fuzzy logic developer to model fuzzy control effectively [17]. For work in this thesis, triangular membership functions were chosen for the fuzzy sets and implemented in MATLAB

The FIS process can be broken into three main steps. Fuzzification, rule generation, and defuzzification.

2.1.1. Fuzzification

Fuzzification is at the beginning of the FIS. It is where the system takes a crisp value (real scaler number) and converts it into a fuzzy linguistic value. For any certain crisp value, a

corresponding fuzzy value can be created as part of different classes of fuzzy membership functions. These membership functions are what define how the crisp value is converted.

There have been a variety of proposed fuzzy membership functions proposed, but according to Zadeh [2] they can all be classified as one of two types: one type consisting of straight lines and one made of curves. Curved membership functions are computationally heavier but perform well for non-linear systems; however, the advantages are limited. Consequently, straight or linear membership functions are more widely used, as they are easy to develop and computationally light.

Triangular fuzzy membership functions are the simplest to develop. Witold Pedrycz [18] and Manu Sharma et al. [16] showed that satisfactory results can often be achieved using triangular membership functions. So initially, this study used triangular membership functions to develop fuzzy set for both a larger and smaller FIS.

As seen in Figure 2, triangular membership functions are defined by two limits and a mode. The lower limit is shown as a, the upper limit is b, and the peak of the triangular membership function is at the mode m. The function is defined over the range a < m < b with a maximum value of one.



Figure 2: Triangular membership function of fuzzy set

2.1.2. Fuzzy Rule Base

The rule base for a fuzzy system is designed with the experience and reasoning of humans. When the relationship between input and output can be defined through basic logic, one can express these relationships as rules in the form of if-then statements (e.g. "if the input is positive and large, then the output is negative and large"). "Positive and large" from the example would be defined by a membership function. The term is not represented by a precise value, but rather it is defined as a range over which an input can be characterized as having partial membership (0-1[100%]) to the term.

Other examples of rules are

- If input 1 is P and input 2 is Z, then output is N
- If input 1 is N and input 2 is N, then output is PL
- If input 1 is Z and input 2 is Z, then output is Z

where zero, positive, negative, and positive-large are abbreviated as Z, P, N, and PL respectively.

2.1.3. Defuzzification

Defuzzification is the process by which a crisp output value is derived from fuzzy parameters. Mitsuishi & Shidama [19] described defuzzification process as converting membership degrees of fuzzy sets to a specific value. There are quite few defuzzification methods available. The most popular methods are the center of gravity method, the center of area method, and the center-average method. For initial fuzzy sets, center of gravity method is used for defuzzification. A crisp value is extracted based on the center of gravity of fuzzy set. Using the center of gravity method, the crisp value z^* is expressed as:

$$z^* = \frac{\int \mu(z). z dz}{\int \mu(z). dz}$$

where z^* is a fuzzy variable, $\mu(z)$ is the area of membership value and z is centroid of the area. The resultant action is divided or distributed into multiple sub areas from different membership function. The resultant area and center of gravity is calculated to find the final

Effectively the equation becomes

$$Crisp \ value = \frac{\Sigma area * centroid \ of \ area}{\Sigma area}$$

2.2. Adaptive Neuro Fuzzy Inference System (ANFIS)

A fuzzy logic controller is useful for designing an intelligent and robust controller and works well on non-linear processes; however, the design process of fuzzy logic controllers is not formalized [20]. Neural networks can be used to tune fuzzy logic set and improve the rule base of a fuzzy controller. This is done with the use of training data [2]. ANFIS utilizes Takagi-Sugeno fuzzy inference system, which is generated and then improved by neural network training. [21] [22]

The ANFIS is one of many methods known as neuro-fuzzy. Adaptive neuro-fuzzy inference system (ANFIS), was first proposed by Jang (1993) [22] and is based on the first-order Takagi-Sugeno fuzzy model. Generally, ANFIS uses either back-propagation or a combination of least square estimation and back-propagation for membership function parameter estimation (Jang and Sun, 1997 [24]). The most important goal of combining fuzzy systems with neural learning capabilities is to implement the robust learning ability of neural networks, which is not part of a regular FIS system. This combination of neural network and FIS allows for the system to learn, improving the performance of the controller. In ANFIS, a Takagi–Sugeno type fuzzy inference system is used to model the system. In Takagi-Sugeno inference systems, the output of each rule is either a linear combination of input variables plus a constant term or only a constant term. The output then consists of a weighted average of every rule's output. This integrated approach, makes ANFIS a universal estimator [25].

Figure 3 shows an ANFIS architecture that was proposed by (Ahmed et al. [26]) that has two inputs x and y and one output f. The rule base contains two Takagi–Sugeno if-then rules as follows:







Rule 1: If x is A_1 and y is B_1 , then $f_1 = p_1x + q_1y + r_1$. Rule 2: If x is A_2 and y is B_2 , then $f_2 = p_2x + q_2y + r_2$ [22]. Figure 3(a) shows the representative fuzzy sets, and Figure 3(b) depicts the training methodology used to change membership functions. Layer 1 is the node function. Layer 2 calculates the product of the node function signals. Layer 3 normalizes the results of Layer 2 by calculating the ratio of each activated rule strength to all activated rules strengths. In layer 4, the output for each node function is calculated, and layer 5 calculates the overall output.

2.2.1. Hybrid Training Method

Hybrid method of training an ANFIS is a combination of least square estimation and back-propagation for membership function parameter estimation [24]. An adaptive network is a multilayer feedforward network where every node performs a particular function referred to as a node function. The node function can vary on each individual node. If a given adaptive network consists of *L* layers and k^{th} layer has #(k) nodes. The node in ith position and k^{th} layer can be denoted by (k,i) and node output can be expressed as O_i^k . A node output is a summation of incoming signals and can be expressed as

$$O_i^k = O_i^k(O_i^{k-1}, \dots O_{\#(k-1)}^{k-1}, a, b, c, \dots)$$
(1)
where *a,b,c*... are parameters related to each node.

If a training data has *P* entries, the error measure E_p , for the p^{th} $(1 \le p \le P)$ entry can be presented as a sum of squared errors, yielding

$$E_p = \sum_{m=1}^{\#(L)} (T_{m,p} - O_{m,p}^L)^2$$
(2)

where $T_{m,p}$ is the *m*th component of *p*th target output vector, and $O_{m,p}^{L}$ is the actual output of *m*th component. To use the gradient decent method for neural network learning, the error rate/gradient is required. The error rate of node (*L*, *i*) can be derived from equation (2) as

$$\frac{\partial E_p}{\partial o_{i,p}^L} = -2(T_{i,p} - O_{i,p}^L) \tag{3}$$

The error rate at the internal node (k,i) can be similarly derived using chain rule:

$$\frac{\partial E_p}{\partial O_{i,p}^k} = \sum_{m=1}^{\#(k+1)} \frac{\partial E_p}{\partial O_{m,p}^{k+1}} \frac{\partial O_{m,p}^{k+1}}{\partial O_{i,p}^k}$$
(4)

Assuming α is a parameter of the network, equation (4) can be expressed as

$$\frac{\partial E_p}{\partial \alpha} = \sum_{0^* \in S} \frac{\partial Ep}{\partial 0^*} \frac{\partial 0^*}{\partial \alpha}$$
(5)

where S is the set of nodes whose output is dependent on α . Taking the derivative of *E* with respect to α yields

$$\frac{\partial E}{\partial \alpha} = \sum_{p=1}^{P} \frac{\partial Ep}{\partial \alpha}$$
(6)

Now $\Delta \alpha$ can be expressed as

$$\Delta \alpha = -\eta \frac{\partial E}{\partial \alpha} \tag{7}$$

where $\boldsymbol{\eta}$ is the learning rate and can be expressed as

$$\eta = \frac{k}{\sqrt{\sum (\frac{\delta E}{\delta \alpha})^2}}$$
(8)

with k as the step size (length of parameter space gradient transition). The k parameter can be used to affect the speed of convergence.

There is two type of hybrid learning: off-line and on-line. In online learning, parameters are updated after each epoch. In this research, off-line batch learning was implemented. This method combines a gradient method and least square estimate (LSE) to identify system parameters.

Assuming the adaptive network has a single output

$$Output = F(\vec{I}, S)$$

where I is input variables set (for this problem only 2 variable), S is parameter set, and F is function implemented by ANFIS. If there exists a composite function, then $H \circ F$ is linear for all values of S, then these values of S can be identified by LSE. If S is decomposed such way that $S_1 \bigoplus S_2 = S$, then

$$H(output) = H \circ F(I, S)$$
(9)

The training data set contains P data pairs. To successfully train the system, P must be greater than than number of linear parameters M. Assuming H(output) will be linear under the elements of S_2 , the equation can be simplified by putting P into (1), yielding

$$\mathbf{A}\mathbf{x} = \mathbf{b} , \qquad (10)$$

where **x** is an unknown vector and S₂ is represented by the elements of **x**. Let $|S_2| = M$, then the dimensions of **A**, **x**, and **b** will be equal to $P \times M$, $M \times 1$ and $P \times 1$. If P > M, equation (10) does not have an exact solution, instead to minimize the squared error of $||A\mathbf{x} - \mathbf{b}||^2$, \mathbf{x}^* , the least squares estimate (LSE) of **x**, is calculated.

The formula used for \mathbf{x}^* is

$$\mathbf{x}^* = (\mathbf{A}^{\mathrm{T}}\mathbf{A})^{-1}\mathbf{A}^{\mathrm{T}}\mathbf{b}$$
(11)

where $(\mathbf{A}^{T}\mathbf{A})^{-1}\mathbf{A}^{T}$ is the pseudo-inverse of **A**.

Finally, the results from the gradient method and the least square estimate are used to update the adaptive network parameters. Each epoch/iteration of neural learning consists of a forward pass and a backward pass. The main goal of the forward pass is to calculate **A** and **b** for each epoch using input data. Subsequently, the parameters in S_2 , which are represented by **x**, are calculated using LSE. With the calculated parameters, the inputs from the training data set are passed through the system to calculate an output. This output is compared with the desired output from the training data set, which is then used for the backward pass using the gradient method to calculate the S_1 parameters. Paremeter of S_1 are related to membership function shape. For this thesis gaussian membership function (Figure 3c) was used, equation for that is

$$\mu_{A^i}(x) = e^{\left(\frac{-(c_i - x)^2}{2\sigma_i^2}\right)}$$

So c_i and σ_i are the parameter in S₁

As S_1 converges, the calculated values of S_2 converges to the global optimum point in the S_2 parameter space. This in turn leads to a decrease the search space dimension in gradient method and results in a faster convergence of the neural training of the fuzzy set.

2.3. Genetic Algorithm

Genetic algorithms are a tool for optimization. Optimization approaches can be classified into two major categories: Classical and Evolutionary. Evolutionary algorithms are based on biological evolution, mainly survival of fittest principle. According to Thomas Back (1996) There are three main types of evolutionary algorithms: Genetic Algorithms (GA), Evolution Strategies, and Evolutionary Programming [27].

Genetic algorithms were developed in the early nineties; Goldberg and Holland were pioneers in introducing GA for use in optimization problems [28]. GA is based on the survival of the fittest phenomena in natural evolution. Here, from a set of random population, a new generation or child population is generated based on the fitness score of the parent's generation. Meaning, the higher the fitness of an individual, the more likely their genes are to be picked for the next generation based on a roulette wheel parent selection. A higher fitness results in a better chance to be picked, but all the members still have a chance to be picked due to the randomness of selection, including the worst (though less likely). For each subsequent generation, the current generation plus some random mutation and crossover are the source of potential genes.

Genetic algorithms are very popular in multi objective optimization problems (MOOP). MOOP deals with a problem that has more than one objective. In the real world, most design problems have multiple objectives. For multi objective problems, the solution is often complex to comprehend with conflicting objectives that require compromises and balance in the solution. In engineering many solutions depend on multiple conflicting objective like improve ride stability and ride safety. Analysis of qualitative and experimental information to find the preferred vector is critical part of finding solution of MOOP. GA is very efficient in searching for the best solution that satisfies all design objectives [29].

2.3.1. GA Operators Initialization

The solution of a genetic algorithm is dependent on the size and variety of the initial population. A random number generator can be used to generate the initial population with an optimal constraint. Use of constraints is not universal in GA, but according to James Baker (1985) an effective initialization approach is to initialize the population close to a known global accepted optimal value for the variable [30]. There are two popular methods for representing the population and gene chromosome. One is to store as binary string and each binary number is a chromosome. Alternatively, one could use real numbers as a chomosome. Although binary representation is computationally light, sometimes it is difficult to comprehend the nature of problem with binary

representation [31]. To make it easiest to implement constraints and work with fitness models, this work used real number representation for the population set.

2.3.2. Fitness Functions and Selection

Fitness function is used to determine the fitness of each design in a given population to facilitate the selection for the next generation in a GA optimization. Fitness function is used to measure the fitness of everyone. A generalized fitness function can be expressed as

$$F(x) = g(f_1(x), f_2(x), \cdots, f_n(x))$$

, where *x* is a variable that is to be optimized, $f_i(x)$ is an objective function, g(x) is a function to combine the values of the objective functions, and *F* is the resulting fitness value.

In this thesis, a rank-based approach was used for fitness, where rank of individual population is used to determine the relative fitness, in addition to the actual fitness value. According to research by James E. Baker, the rank-based approach helps to improve convergence [30].

Selection is the process that decided scope of reproduction for each individual population. Selection is comprised of two different steps. First, a fitness value for each individual is converted into the probability that an individual is selected for reproduction. Second, sampling is performed, where individuals are picked for reproduction based on comparative probability with other members of population pool. Bias, spread, and efficiency are three key parameter that determines the performance of algorithm. Bias is the absolute difference between the actual and expected probability of getting selected for reproduction. Spread is the range of time one can be picked, efficiency is the execution time for the algoritm.

Roulette wheel mechanism and stochastic uniform sampling are the two most popular methods for selection. Prior to both methods, the probability of being chosen is calculated based on the fitness values of the polulation. The probability is then used to map all members of the parent population to ranges of values from 0-1. Using the rank-based approach, the members are arranged such that the member with the highest probability has a range that begins at the start of the 0-1 mapping. All members of the population are then mapped in descending order ending with the least fit member with a range that ends at the value of 1. During roulette wheel sampling, a uniform random variable from 0 to 1 is generated and used to select individuals based on the rankbased mapping. In this method, it is possible to select the same parent member twice. With stochastic uniform sampling (SUS) uses a linearly distributed set of values from 0-1 to choose. SUS is a recombination technique and is used to pass and recombine a potentially useful solution from one generation to the next; according to Jayabal et al [32] with stochastic uniform sampling, the algorithm moves linearly, and each parent gets equal importance. This reduces the chance of having many copies of the fittest member, and guarantees that at least one copy of the fittest member will be copied. For these reasons, stochastic uniform sampling was used in this thesis.

2.3.3. Crossover

Crossover is the main method of producing a new generation offspring in a genetic optimization. Under crossover new offspring are created where they inherit some chromosome features from both parents. Single point crossover is the most popular crossover method. In single point crossover, a point is randomly selected from a list of pre-specified points on the chromosome; all information prior to that point comes from the first parent and all remaining information is provided by the second parent, creating an offspring chromosome. In the multipoint method randomly, multiple crossover points are picked and sorted in ascending order. Figure 4 shows the

use of two successive crossover points, where the parent source of the chomosome switches when the offspring is created. In this research, the multipoint crossover method was used.



Figure 4: Scattered Crossover

2.3.4. Mutation

Mutation in a genetic algorithm allows for small random changes to individuals. Mutation generally improves diversity and broadens the search space of GA beyond the traits that were present in the initial population.

The natural theory of evolution accepts that organisms will diversify in order to survive, and mutation is the method to achieve additional diversity. The most common form of mutation is completely random, but a better method is to have mutation occur in response to specific stresses. The mutation will be more beneficial to direct offspring and be specific to the given stress. To this aim, John cairns pioneered the idea of adaptive mutation [33].

Adaptive feasible mutation is a process in a generic algorithm where more successful genes are less likely to be mutated and least significant bits are more likely to be mutated to increase the fitness and accuracy. Under this method of mutation, the chance of changing high fitness chromosomes is decreased and low fitness chromosomes are more likely to be altered as shown in Lebelli et el [34]. Given the additional benefits, adaptive feasible mutation was used in this thesis.

3. Methodology

3.1. Mathematical Model for System



Figure 5: Process flow for suspension model

Figure 5 illustrates the model of the suspension system and control system. For the purposes of this thesis, a quarter car model was used. The design goal for this system is to stabilize and improve the response of the suspension. This is accomplished by measuring the velocity and position of the quarter-car mass, using them as inputs to a sequential fuzzy logic set to calculate the force used to dampen the suspension. Overall, the system is modeled as a second-order differential equation of forced damped vibration. The equation for current quarter car model can be written as shown in equations (12) and (13).

$$m_1 \ddot{x}_1 = -b_1 (\dot{x}_1 - \dot{x}_2) - k_1 (x_1 - x_2) + u \tag{12}$$

$$m_2 \ddot{x}_2 = b_1 (\dot{x}_1 - \dot{x}_2) + k_1 (x_1 - x_2) + b_2 (\dot{w} - \dot{x}_2) + k_2 (w - x_2) - u$$
(13)

As per Dorf, Richard C., and Robert H. Bishop [35], equation (12) and (13) can be transformed into state-space equations shown in equations (14) and (15). A state-space model gives easier accessibility to multiple variables and is easily modelled in Simulink.

$$\dot{\boldsymbol{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}\mathbf{u} \tag{14}$$

$$\mathbf{y} = \mathbf{C}\mathbf{x} + \mathbf{D}\mathbf{u} \tag{15}$$

Here,

State Vector,
$$\mathbf{x} = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$
 where,
 $x_1 = x_1$
Input signals, $\mathbf{u} = \begin{bmatrix} w \\ \dot{w} \\ u \end{bmatrix}$ $x_2 = \dot{x}_1$
 $x_3 = x_2$
 $x_4 = \dot{x}_2$

Now arranging the equation of $\dot{x}_1, x_1, \dot{x}_2, \ddot{x}_2$ in terms of x_1, x_2, x_3, x_4, u, w and \dot{w} yields:

$$\dot{x}_1 = x_2 \tag{16}$$

$$\dot{x}_3 = x_4 \tag{17}$$

$$\dot{x}_{2} = \ddot{x}_{1} = \frac{-b_{1}(\dot{x}_{1} - \dot{x}_{2}) - K_{1}(x_{1} - x_{2}) + u}{m_{1}}$$
So, $\dot{x}_{2} = -\frac{b_{1}}{m_{1}}(x_{2} - x_{4}) - \frac{k_{1}}{m_{1}}(x_{1} - x_{3}) + \frac{u}{m_{1}}$

$$\dot{x}_{4} = \ddot{x}_{2} = \frac{b_{1}(\dot{x}_{1} - \dot{x}_{2}) + K_{1}(x_{1} - x_{2}) + b_{2}(\dot{w} - \dot{x}_{2}) + K_{2}(w - x_{2}) - u}{m_{2}}$$
(18)

Then,
$$\dot{x}_4 = \frac{b_1}{m_2}(x_2 - x_4) + \frac{k_1}{m_2}(x_1 - x_3) + \frac{b_2}{m_2}(\dot{w} - x_4) + \frac{k_2}{m_2}(w - x_3) - \frac{u}{m_2}$$
 (19)

From equation (16) to (19),

Sequential Fuzzy Logic is used to calculate the commanded force in the model. The model uses vertical velocity and displacement of suspension as inputs to calculate actuator force. For fuzzy logic, a Mamdani Type Fuzzy Interference set was used [36], because Mamdani set rules can be developed from human experience in an easy to use linguistic form.

3.1.1. Quarter Car Model

The modeling parameters for the quarter car are similar to those in the work of Alleyne, Andrew, and Rui Liu[37] and modified within practical limits. Initially, the data was as shown in Table 1.

Quarter Car	Tire and Wheel
$m_1 = 250 \text{ kg}$	$m_2 = 25 \text{ kg}$
$b_1 = 1,500 \text{ N/ms}$	$b_2 = 600 \text{ N/ms}$
$K_1 = 15,000 \text{ N/m}$	K ₂ = 200,000 N/m

Table 1: Quarter Car Properties

The system was simulated using Matlab and Simulink. The Simulink model consists of a quarter-car model and a fuzzy logic controller. The input for the fuzzy logic system's rule set is the suspension deflection and the velocity. The main goal of the system is to minimize the vibration by controlling the rate of change in the suspension deflection.

To test the controller effectiveness, it was simulated with a disturbance input of a regular bump in the road, a large pothole, and a rough road profile. Each condition was tested independently and were combined for a final test.

The bumps in the road were simulated using two-different step functions, each lasting one second. The peak of the first step function was 6 cm and the second one was 10 cm (Figure 6a). For a large pothole, a sinewave of 6 cm amplitude and 1 rad/sec frequency was used, that lasted for 5.13 seconds (Figure 6b). To simulate the continuous disturbance of a rough road, a sine wave of 2 cm amplitude and frequency of 15 rad/sec was present throughout the simulation runtime (Figure 7a).

To accommodate various disturbances, the input fuzzy logic set was adjusted to give an appropriate range based on the movement range and limitations of the physical system. The output corresponding to the applicator force was selected with the goals of stability and practicality. Based upon the expected disturbances from the road, physical parameters of the system (spring and dampening coefficients) were adjusted.







A road bump is simulated by a step function (Figure 6a) to represent a sudden obstacle and associated rise and fall vertically. Figure 6b depicts the disturbance of a pothole, which is simulated with a portion of a high-amplitude sine wave. A continuous sine wave with low amplitude was picked to model a rough road as shown in Figure 7a. Finally, a combined disturbance was used (Figure 7b) as the worst scenario for tuning the initial fuzzy system.





Figure 7 (b): Combined Road Disturbance

Figure 7b shows the combined disturbance, which is simulated for a duration of 35 seconds. In this combined disturbance, the rough road signal was present for the entire 35 seconds, the road bump signal was present for 1 second starting at 6 and 10 seconds of simulation time, and the pothole signal was present between 20 to 25 seconds of simulation time.



3.2. Fuzzy Inference System (FIS) Development

Figure 8: Steps in development of system

Fuzzy inference systems (FIS) are also known as fuzzy-rule-based systems, fuzzy models,

or fuzzy controllers when used as controllers. An FIS consists of five parts as seen in Figure 9a [22]:

- i. Rule Base contains fuzzy rules in if-then format
- ii. Database contains membership function information including shape and value
- iii. Decision Making Unit performs the rule inference operation
- iv. Fuzzification Inference Unit converts crisp input into fuzzy linguistic parameter

v. Defuzzification Inference Unit - converts fuzzy value into crisp output



Figure 9 (a): Block diagram of Fuzzy Inference System



Figure 9 (b): Block diagram for sequential fuzzy decision

Figure 9a shows a block diagram of an FIS, and Figure 9b shows a system comprised of two sequential FIS.

3.2.1. Development of Sequential Fuzzy Logic

For the fuzzy logic controller, two different fuzzy sets were used to form a sequential fuzzy system. The first fuzzy set covered most cases and worked for a large range. The second fuzzy set worked when the magnitude of vibration attenuated to a lower range. This improved control of the system at important phases, allowing rules to be more compartmentalized and easier to work with. Both fuzzy sets take vertical velocity and displacement of the car as inputs and output the desired actuator force to control the vibration. Triangular membership functions were used for velocity, displacement, and force. For larger primary fuzzy set, five input membership functions were used for force. This gave the system sufficient freedom in controlling the vibration. The range for the parameters is shown in Table 2 and the rule matrix is shown in Table 3.

Table 2: Larger Range Fuzzy Set Parameters

	Maximum Value	Minimum value
Vertical velocity of CAR	2 m/s	-2 m/s
Vertical displacement	5 cm	-5 cm
Actuator force	2500 N	-2500 N

The parameter values used for velocity and displacement are based on Sharma et al [16] but the parameter for the actuator is based on an industrial linear actuator with push load of 2500N [38], the reason for picking this actuator parameter is size and force capability. An actuator with a higher force capability would have been better to control the vibration but too bulky to use in a standard car suspension. The rule matrix for the larger fuzzy set is shown in Table 3.

Disp	NL	Ν	Ζ	Р	PL
NL	PLL	PLL	PL	Р	Ζ
Ν	PLL	PL	Р	Ζ	Ν
Ζ	PL	Р	Ζ	Ν	NL
Р	Р	Ζ	Ν	NL	NLL
PL	Ζ	Ν	NL	NLL	NLL

Table 3: Larger fuzzy membership table

The smaller fuzzy set was used to provide finer control in the central portion of larger fuzzy set and have a smaller range of parameters as seen in Table 4. The design of the smaller fuzzy set was more focused and decision about the parameters were made using trial and error method, observing which values performed best. The chosen rule matrix for the smaller fuzzy set is shown in Table 5.

 Table 4: Smaller Range Fuzzy set parameters

	Maximum Value	Minimum value
Vertical velocity of CAR	0.5 m/s	-0.5 m/s
Vertical displacement	1 cm	1 cm
Actuator force	800 N	-800 N

 Table 5: Smaller fuzzy membership table

Disp	Ν	Ζ	Р
Ν	PL	Р	Ζ
Ζ	Р	Ζ	Ν
Р	Ζ	Ν	NL

The resultant surface plots for both fuzzy sets are shown in Figure 9. It can be seen that the the larger is less linear, which allows it to compensate for different type of disturbance in ranges near the limits of the actuator. The smaller fuzzy set is more linear in nature but still applies a higher gain for small imputs.





b

Figure 10: Surface Plot for (a) Larger Fuzzy Set, (b) Smaller Fuzzy Set

3.3. Adaptive Neuro Fuzzy Inference System (ANFIS) Development

For an ANFIS, offline training is performed using the same data that used to develop Mamdani FIS and is used to compare the performance change between FIS and ANFIS system. and were tuned using Neural learning. Given the difficult in designing with gaussian membership functions, they are often not chosen; however, given the advantages of using a neural network, the shape of the membership functions were changed from triangular to gaussian prior to being tuned by the neural network.

Table 6 shows the training data used, providing both the input and and the desired output . This data was used by the Neuro-Fuzzy Design tool in MATLAB to tune the membership functions.

Velocity	Displacement	Force
(m/s)	(m)	(N)
-2	-0.05	1200
-0.6	-0.05	1000
-2	-0.0125	1000
-0.6	-0.0125	800
0	-0.0125	700
-0.6	0	700
0	0	0
0.6	0	-700
0	0.0125	-700
0.6	0.0125	-800
2	0.0125	-1000
0.6	0.05	-1000
2	0.05	-1200

Table 6: Training data for Neuro-Fuzzy System.

For this study a combined hybrid model using both LSE and gradient descent methods (as described earlier) was used to train the fuzzy system. The resultant rule set increased the system performance within the restriction of parameters. Below in Figure 11, the change of membership function due to neural learning is shown. The training error for each epoch/iteration of the training is shown in Figure 12. It can be seen that the system converged after 3 epochs using the hybrid method compared to back-propogation method, which failed to converge after 1000 epochs. Clearly, the Hybrid Neural Learning method converged faster resulting in a low error of 0.0013 after 10 epoch whereas with back propagation method maintained a high error of 872.61 after 1000 epochs.



Figure 11 (a): Membership function for displacement before Neural Training



Figure 11 (b): Membership function for displacement after Neural Training



Figure 12 (a): Training error using Hybrid Neural Learning



Figure 12(b): Training error using back-propagation neural learning

Figure 13 shows a comparison of membership functions between regular FIS and ANFIS. In addition to changes in the membership function shape, there were also changes to the range. The changes in the membership functions can also be seen in Figure 14, which depicts the FIS surface before and after training with a neural network. The main difference between the two is the high gain (shown as a steep surface) in the middle associated with the displacement input.









с

Neuro Fuzzy









and Neuro Fuzzy System.

- a) Input displacement (Fuzzy)
- b) Input velocity (Fuzzy)
- c) Output force (Fuzzy)
- d) Input displacement (Neuro Fuzzy)
- e) Input velocity (Neuro Fuzzy)







b

Figure 14: (a) Surface Picture of regular FIS (b) Surface of FIS trained with Neural Learning

3.4. Optimization Through Genetic Algorithms

To further improve the system, the damper and spring combinations were modified to provide improved performance for the car. This was done using a genetic algorithm. For this optimization problem, the damping co-efficient the limited to a range of 750-1800 and the spring constant was limited to a range of 12,000-18,000.

To begin, an initial random population was created within the limits. The size of the population was 50, and each member of the population is defined with two variables. For fitness function squared sum of force, F and displacement, X1 was taken at every 0.01 sec. So, for total 15 sec the data point was n=15/0.01=1500. Total number of peaks, p was calculated from dampned displacement as shown in Figure 15.



Figure 15: Number of Peaks, p

The fitness function used is

$$F = \frac{1}{n} \sqrt{\sum_{k=1}^{n} (F_k * X_k)^2} * (p - C)$$

where $\frac{1}{n}\sqrt{\sum_{k=1}^{n}(F_k * X_k)^2}$ is the RMS (Root Mean Square) of the work on the quarter-car mass, *p* is the number of peaks in the vibration, and *C* is a constant chosen based on the number of expected oscilations. For ride stability, three parameters were considered: the amount of displacement, force applied, and frequency. Force and displacement is combined into actuator work. Works done is related to the performance of the actuator, which in turns is related to stability of ride and logivity of suspension system. So, one main goal was to minimize the work done by the actuator but keep the vibration dampened. Additionally, the frequency of vibration is also related to ride comfort, so lower frequency was preferred to higher, and it was seen that the number of waves in different scenario varies between 36-48. To penalize the high frequency vibration, a constant value (C=35) was used and subtracted from the number of waves and the difference was multiplied with RMS value of work done. With this fitness function, all criteria which contribute to the desired performace are accounted for and the genetic algorithm will work to minimize the value, finding the best balance.

In the genetic algorithm optimization, the crossover rate for the new generation was 0.8 and Mutation rate was 0.01. For crossover scattered function was used. This is a built in Matlab function where the child is created by taking a random part form each parent. Scattered creates a random binary vector. Scatter crossover select gene from first parent when binary value of random crossover vector 1 and gene from second parent if binary value is 0. For example: Lets take two individual with values of (1500, 15000) and (1200, 17000). If considering the first variable (dampening coeficiente) of the two individuals, 1500 and 1200, they can be presented in binary as

parent1(1500) = [10111011100]

parent2(1200) = [10010110000]

If the given random crossover vector is $[1\ 1\ 0\ 0\ 1\ 0\ 0\ 1\ 1\ 0]$ the resultant child value would be child = [10011110100], which is decimal 1268.

For mutation, the adaptive feasible function is used to keep the resultant child within the range specified. The adaptive feasible function randomly generates directions that are adaptive with respect to the last successful or unsuccessful generation. A built-in Matlab function is used for the adaptive mutation, where a step length was chosen to progress the evolution in each direction so that bounds are satisfied.

4. Results & Discussion

FIS system:

Initially, a sequential FIS model was used to calculate the force for the actuator. As seen in Figure 16, the overall response of the system against high frequency low amplitude wave (rough road) was not satisfactory as the displacement remains attenuated to a large extent; however, the actuator with FIS controller did perform well with the road bump at 6 seconds. Further comparison of the system performance can be seen in Figure 17, which compares the performance of the system with and without the actuator. The figure shows that the system with the actuator performed better than the basic spring damper system without an actuator.



Figure 16: Disturbance vs displacement of quarter car mass with FIS controlled actuator



Figure 17: Displacement of QCM mass with FIS controlled actuator and without any actuator

Under FIS the system was damping vibration generated by sudden bump, but in scenario with bump with larger disturbance which is out of preset parameter of fuzzy set; the force was inadequate to control the damping. Also, we can see controller fails to make adjustment when the larger disturbance that is out of range was applied. There was no controlling force (Figure 18) between 10-12 sec, this is due to disturbance is out of range for fuzzy set and the controller fails to adjust accordingly. Also, even though system was able to control the continuous disturbance of high frequency, to a degree the result was not good enough.



Figure 18: Force under FIS system

ANFIS performance:

With new rule base generated with Neural learning, the dampened vibration has comparatively lower amplitude and overall improved performance as shown in Figure 18. In Figure 19, it can be seen that the force applied is comparatively lower with ANFIS model, the only exception is when there is larger disturbance that is out of our specified limit. Consequently, the ANFIS model can adapt and apply a higher force gain to control the vibration more effectively in this scenario. This can be seen in Figure 19 that even when vertical displacement is out of range between 10-12 sec, ANFIS model can infer higher force (Figure 20) and by applying that force, vibration is getting damped better compared to FIS model.



Figure 19: Comparison of Displacement with ANFIS and with FIS



Figure 20: Comparison of Force applied with ANFIS and with FIS

ANFIS with Optimized with Genetic Algorithm (GA) Parameter:

The GA used an initial population of 50, and the optimization was set to run for 100 generations with stalling criteria set at 5 generations. With multiple run the solution always converged between 12-14 generation, example from the best run is shown in Figure 21.



Figure 21: Convergence of Optimization

Even the solution was set to run for 100 generations, stalling criteria with function tolerance of 1e-

6, which triggered an early termination as shown in Figure 22.



Figure 22: Stopping criteria for GA optimization

The result from three different runs are shown in Table 7. All the results are close in magnitude. The results shown are from the best optimization run.

Run	No of generation	b1	k1	fitness
1	13	812.602	12168.59	1.01506
2	12	845.49	12346.79	1.0338
3	14	822.961	12297.37	1.01325

Table 7: Results from GA Optimizations

From random run of 100 different samples of b1 and k1, the best value was b1= 1062.95 and k1=12372.408, resulting in Fitness = 1.12911. The fitness values for 100 random runs is shown in Figure 23 (all the data of random run is in appendix B). The fitness values from all 100 radomly generated systems (set of *b* and *k*) were worse (higher) than every GA run. From Table 7 and Figure 23, it is evident that genetic algorithm optimization provided an improved fitness value by 11.4% compared to best result from random run. The optimization tended to converge within 13 generations, ending with the stall criteria (one exception was 22 generation). For all the optimization runs the final fitness value was always comparable, highest and lowest valued best fitness was within 2% of each other.



Figure 23: Fitness value from 100 random run



Figure 24: Displacement comparison between GA optimized ANFIS and ANFIS

Figure 24 shows that the relative performance of the system after the optimization of the spring and damper. It can be seen that optimized damper and spring value perform better in bump,

but everywhere else damping performance of regular damper and spring under ANFIS is unchanged.



Figure 25: Force comparison between GA ANFIS and ANFIS

Figure 25 shows a comparison of the force used by the actuator before and after the optimization. The force required for the optimized ANFIS is consistently lower during rough road period. To better show the performance difference Figure 26 focusses on the results from 5 to 12 seconds; the amplitude of displacement under optimized physical parameter is clearly lower than the displacement with the original parameters.



Figure 26: Displacement comparison between GA ANFIS and ANFIS time 5 to 12 sec



Figure 27: Force comparison between GA optimized ANFIS and FIS

Finally, the force utilization of FIS system and ANFIS system with GA optimization is compared in Figure 27. It can be seen that the optimized ANFIS system utilizes much less force, which is beneficial for actuator health, performance, and energy usage. It can be concluded that the FIS system with actuator control improved performance of the basic spring damper system, but basic FIS system design methods have limitations which can be overcome by using a neural network to train the membership function. Finally using GA optimization for spring and dampner can further improve the response of the system to a forced vibration with lower force required for the actuator.

5. Conclusion

In this thesis, an active vibration control method was developed for automotive suspension. The model was tested against a quarter car model that was represented using a state-space model. The force required to control the vibration actively was calculated using FIS model. The result was inadequate for few cases, so the system was further improved by the development of an adaptive neuro fuzzy model. The ANFIS model was trained with a hybrid of least square and the gradient method. The training with the hybrid approach converged very fast within few epochs and error was within the order of 1/1000. When only training with regular back propagation, the convergence was slow, and error was high. For example, at 1000 epoch the error was about at 872 with back propagation.

To further improve results, a genetic algorithm was used to optimize the model parameters of the system. A bounded, continuous search space was defined for the critical parameters of damping coefficient and spring constant of the suspension. The fitness function was based on actuator energy and frequency of vibration. Finally, the optimization result was compared with a random search space to verify the effectiveness of GA. It is found that result from GA yielded a better fitness compared to best result of random runs. That indicates GA optimization was successful in finding a solution.

In future work, instead of a quarter car model, a whole car model can be used to test more accurately and to reflect the complexity of a whole car suspension. Additionally, optimizing the basic spring damper system before using actuator with either of FIS or ANFIS controller may produce different results. The GA optimization population sample can be increased along with stall criteria to run the GA optimization longer and see if there is further improvement of result. The GA could also be used to optimize additional variables, including the fuzzy controller. The system was simulated using Simulink to determine the fitness; this is computationally heavy. Alternative methods of calculating the fitness can be explored. Additionally, rather than using the generated disturbances used in testing, real road data can be used as an alternative to more accurately reflect real-life road conditions. Finally, whole model can be experimentally tested for further improvement, main challenge there will be the actuator, even though comparable actuator with required weight and force capacity is already available, their reaction time is slow for an effective active vibration control; however, it can be inferred from the results that the proposed ANFIS model for vibration control is effective and GA optimization provides further control performance within practical search space for physical parameters.

To summarize, an active vibration control for suspension was studied using a quarter car model. Initially FIS controller is used to calculate force required for actuator, which improved the performance for a road bump, but struggled in continuous rough road disturbances. Then ANFIS was used to train and modify the membership functions, leading to substantial increase in performance with both rough road and bump disturbances. Finally, the basic suspension was optimized using GA, which led to additional performance gains.

Appendices

Appendices A: Sample calculation of a Fuzzy Inference system

Let's a consider the FIS system used in this thesis. The system has 2 input:

Distance (5 level): NL, N, Z, P, PL ; Speed (5 level): NL, N, Z, P, PL

1 output:

Force (7 level): NLL, NL, N, Z, P, PL, PLL

Few examples of rules:

If (Velocity is Z) and (Displacement is Z) then (Force is Z)

If (Velocity is Z) and (Displacement is NL) then (Force is PL)

Now For a single test case of velocity (V) of -0.2 m/s and displacement (D) of -4 cm first the input V and D are fuzzified.

Fuzzification:

Here, by definition and according to figure 28, V is subset of both N and Z ; D is subset of both

NL and N.



Figure 28: Fuzzy Input and outputs

Rule base:

Now for fuzzy values of V and D 4 different rule is triggered and each results a different fuzzy output.

Rule triggered in this case are:

- If (Velocity is Z) and (Displacement is NL) then (Force is PL)
- If (Velocity is N) and (Displacement is NL) then (Force is PLL)
- If (Velocity is Z) and (Displacement is N) then (Force is P)
- If (Velocity is N) and (Displacement is N) then (Force is PL)

It can be seen in last blue column of figure 11.

Defuzzification:

Defuzzifications first combines all the fuzzy actions into one action. Then transform said action

into a crisp vale. As can be seen in figure 29.



Figure 29: Surface Plot for (a) Combined output area, (b) Combined output with COG (red mark) In this work center of gravity (COG) is used to calculate the crisp value. the resultant crisp force is 1366 N.

Appendices B:

Sample No	Damping coefficient, b1	Spring constant, k1	Fitness value
1	1307.349	14941.703	1.95361
2	1241.608	13283.9	1.63453
3	1279.535	12238.746	1.52045
4	957.692	12346.285	1.39405
5	822.849	18175.934	1.66977
6	1271.121	14378.246	1.45121
7	1462.667	12258.346	1.91437
8	1079.058	14861.688	1.75343
9	1324.138	17963.999	2.49711
10	890.154	15673.233	2.17602
11	1084.19	16847.348	1.8603
12	1282.024	17660.15	2.46784
13	957.019	17097.77	1.90256
14	1241.324	13219.588	1.59165
15	839.248	15737.73	2.2137
16	1469.808	14552.893	2.29201
17	1188.951	13678.59	1.68399
18	1313.45	13913.213	1.7723
19	1129.468	17243.075	1.94396
20	1418.177	19194.686	2.66134
21	1160.412	13039.683	1.56551
22	861.971	13931.312	1.73081
23	1380.538	13907.116	2.14873
24	1360.714	13826.437	2.11868
25	1446.948	14624.878	2.29214
26	897.446	13883.129	1.69174
27	1212.034	15549.666	1.64254
28	1013.745	18231.215	2.14921
29	1188.948	16122.927	1.74654
30	1437.895	14143.793	2.22237
31	1317.9	17652.968	2.42623
32	1035.334	16258.662	1.72754
33	806.891	12404.626	1.28575
34	1148.098	17843.754	2.03894
35	1450.508	12974.297	2.01244
36	1176.618	15520.43	1.63649
37	758.927	14528.42	1.90432

Table 8: Fitness data from 100 randomly generated damping co-efficient and spring constant

			1
38	871.637	17957.134	1.58871
39	983.411	15963.999	1.99483
40	874.237	16514.865	1.64241
41	947.228	16905.593	1.83604
42	1266.911	17611.137	2.47218
43	1087.906	12628.66	1.46228
44	921.733	18850.03	2.3783
45	864.284	18193.627	1.64653
46	1153.757	19471.01	2.88902
47	808.632	15320.087	2.07156
48	829.99	19214.236	1.90974
49	753.476	17811.828	2.1766
50	1362.977	18515.21	2.57118
51	813.327	14998.37	1.99625
52	944.903	18000.514	2.11909
53	1073.56	18829.857	2.25746
54	886.385	13978.522	1.80245
55	859.154	13020.514	1.67709
56	1401.969	16347.784	2.62882
57	1162.395	13087.161	1.58856
58	1389.773	16665.413	2.71153
59	1013.214	15849.372	2.0493
60	1051.356	12569.75	1.4207
61	929.937	12924.892	1.69166
62	887.931	13799.644	1.72984
63	1062.95	12372.408	1.12911
64	1427.037	19085.904	2.63508
65	1118.148	15669.395	1.6358
66	1003.29	18750.404	2.28589
67	1026.935	12834.021	1.96021
68	1335.189	14923.041	1.93281
69	931.268	15029.341	2.2648
70	822.341	12989.8	1.34096
71	1456.538	19171.009	2.62999
72	1181.406	12448.347	1.50154
73	926.085	14648.689	2.35638
74	1365.896	12115.526	1.78855
75	782.268	13267.425	1.59915
76	1236.837	17487.918	2.43018
77	1235.809	15381.928	1.60796
78	1160.257	14222.406	1.73239
79	1308.52	13417.163	1.34156
80	1265.082	13376.334	1.54591

81	1026.363	16692.139	1.82575
82	1335.171	12608.443	1.8647
83	1447.039	17817.845	2.89585
84	1115.094	15268.939	1.56367
85	1085.088	14297.621	2.32937
86	1131.381	15830.787	1.66328
87	1363.221	17961.236	2.4854
88	1233.239	14839.57	1.53767
89	1358.685	15996.192	2.12198
90	1013.045	19042.512	2.36616
91	1406.957	16126.173	2.59157
92	1216.856	16402.835	1.78004
93	905.807	14259.347	1.8194
94	1103.193	13728.661	1.65614
95	1383.232	13460.732	2.06519
96	919.441	13280.31	1.93904
97	920.748	15267.74	2.39168
98	983.327	18925.347	2.34186
99	1072.656	13386.122	1.89297
100	1428.661	19348.113	2.66395
Best			
(Fitness)	1062.95	12372.408	1.12911
Worst			
(Fitness)	1447.039	17817.85	2.89585
Median			
(Fitness)	829.99	19214.24	1.90974

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